INTERNATIONAL **STANDARD**

ISO 3601-1

Fifth edition 2012-03-01 **AMENDMENT 1** 2019-09

Fluid power systems -

Part 1:

Inside diameters, cross-sections, tolerances and designation codes

AMENDMENT 1

Transmissions hydrauliques et pneumatiques — Joints toriques —

Partie 1: Diamètres intérieurs, sections, tolérances et codes AENDE.

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AMENDEMENT 1





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This document was prepared by Technical Committee ISO/TC 131, *Fluid power systems*, Subcommittee SC 7, *Sealing devices*.

A list of all parts in the ISO 3601 series can be found on the ISO website.

Any feedback or questions on this document should be directed to the user's national standards body. A complete listing of these bodies can be found at www.iso.org/members.html.

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Fluid power systems — 0-rings —

Part 1:

Inside diameters, cross-sections, tolerances and 1.2012 Amd 1.2019 designation codes

AMENDMENT 1

Page 1, Scope

Replace the Scope with the following:

This document specifies the inside diameters, cross-sections, tolerances and designation codes for O-rings used in fluid power systems for general industrial and aerospace applications.

The ISO 3601 series of standards basically addresses 0-rings with moulded cross-sections without a radial joint. The dimensions and tolerances specified in this document are suitable for any elastomeric material, provided that suitable tooling is available.

The tooling most commonly available is based on 70 IRHD NBR shrinkage rates (see ISO 48). For materials that shrink differently from this standard NBR compound, a special mould can be required to maintain the mean diameters and the tolerance limits listed.

Page 31, Table A.1

Replace <u>Table A.1</u> with the following (three tolerance ranges for non-standard O-rings with cross section diameters $d_2 > 8.4$ mm have been added):

 ${\it Table A.1-Tolerances\ of\ cross-section\ diameters\ for\ non-standard\ O-rings }$

	Cross-section	Tolerance	Cross-section	Tolerance
	d_2		d_2	
	mm	mm	in	in
	$0.80 \le d_2 \le 3.15^a$	±0,08	$0,031 \le d_2 \le 0,124^a$	±0,003
	$0.80 \le d_2 \le 2.25^{\text{b}}$	±0,08	$0.031 \le d_2 \le 0.089$ ^b	±0,003
	2,2 5 < d ₂ ≤ 3,15 ^b	±0,09	$0,089 < d_2 \le 0,124^{\text{b}}$	±0,004°
	$3,15 < d_2 \le 4,50$	±0,10	$0,124 < d_2 \le 0,177$	±0,004 ^c
ć	$4,50 < d_2 \le 6,30$	±0,13	$0,177 < d_2 \le 0,248$	±0,005
•	$6,30 < d_2 \le 8,40$	±0,15	$0,248 < d_2 \le 0,331$	±0,006
	$8,40 < d_2 \le 10,00^{\text{d}}$	±0,20	$0,331 < d_2 \le 0,394^{d}$	±0,008
	$10,00 < d_2 \le 12,00^{\rm d}$	±0,25	$0,394 < d_2 \le 0,472^{d}$	±0,010

Applies to class A only.

Applies to class B only.

Differences between tolerance values are due to conversion of dimensions from metric to inch and rounding rules.

Tolerances apply to O-rings with inside diameters $d_1 \le 660$ mm (25,98 in). Tolerances for diameters $d_1 > 660$ mm (25,98 in) shall be agreed upon between customer and manufacturer.

Table A.1 (continued)

Cross-section d_2	Tolerance	Cross-section d_2	Tolerance
mm	mm	in	in
$12,00 < d_2 \le 14,00^{d}$	±0,28	$0,472 < d_2 \le 0,551^{d}$	±0,011

Applies to class A only.

2

Applies to class B only.

Differences between tolerance values are due to conversion of dimensions from metric to inch and rounding rules.

Tolerances apply to 0-rings with inside diameters $d_1 \le 660$ mm (25,98 in). Tolerances for diameters $d_1 > 660$ mm (25,98 in) shall be agreed upon between customer and manufacturer.