

**ASME B18.2.3.4M-2001**  
[Revision of ANSI/ASME B18.2.3.4M-1984 (R1995)]

# METRIC HEX FLANGE SCREWS

AN AMERICAN NATIONAL STANDARD



The American Society of  
Mechanical Engineers



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**[Revision of ANSI/ASME B18.2.3.4M-1984 (R1995)]**

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## FOREWORD

American National Standards Committee B18 for the standardization of bolts, screws, nuts, rivets, and similar fasteners was organized in March 1922 as Sectional Committee B18 under the aegis of the American Engineering Standards Committee (later the American Standards Association, then the United States of America Standards Institute and, as of October 6, 1969, the American National Standards Institute, Inc.), with the Society of Automotive Engineers and the American Society of Mechanical Engineers as joint sponsors. Subcommittee 2 was subsequently established and charged with the responsibility for technical content of standards covering wrench head bolts and nuts.

At its meeting on December 4, 1974, Committee B18 authorized preparation of a series of standards for metric fasteners. Subcommittee 2 was assigned responsibility for developing standards for metric hex bolts, screws, and nuts.

At a meeting on September 22, 1976, Subcommittee 2 organized the contents of a standard covering eight different hex head screw and bolt products. Actual drafting was postponed until ISO/TC2 could reach final decisions relating to basic dimensions and characteristics of hex bolts, screws, and nuts. At ISO/TC2 meetings held in April 1977, final actions were taken. Committee B18 affirmed the TC2 decisions at a meeting on June 29, 1977, and drafting of this Standard was started.

In February 1978, Committee B18 established a cooperative program with the Department of Defense to draft American National Standards for metric fasteners in such a way that they could be used directly by the government for procurement purposes. The Department of Defense requested that each of the eight products be covered in separate standards, and Subcommittee 2 accepted this approach at its meeting on June 27, 1978. The standard was approved by the secretariat and submitted to the American National Standards Institute for designation as an American National Standard. This was granted on May 17, 1979.

ANSI B18.2.3.4M-1979 was introduced to ISO/TC2 by the United States of America with the recommendation that it be considered as the basis for development of ISO standards for metric hex flange screws. At the March 1980 ISO/TC2 meetings, the Canadian delegation was invited to accept responsibility for an analytical research study to examine the relationships between wrenchability, head material volume, head proportioning, and manufacturing convenience. Stelco, Inc., conducted the study with E. M. Alexander supervising.

Following these ISO/TC2 meetings, the Canadian research was reviewed, and based on the findings, designs of metric hex flange and metric heavy hex flange screws were established. Preparation of ISO standards for both series of flange screws was authorized.

Following these ISO/TC2 decisions, B18 subcommittee 2 initiated this revision of ANSI B18.2.3.4M-1979 with the purpose being to bring it into conformance with the proposed ISO standards. Relatively few changes were necessary; the principal ones included:

- (a) discontinuation of size M20
- (b) modest adjustments in values for flange diameters bearing circle diameters, head heights, and wrenching heights
- (c) slight change in width across corners tolerances
- (d) a more effective and simplified technique for gaging the head

This revised Standard is in essential accord with ISO standards now under development.

This revision was approved by letter ballot of Subcommittee 2. Following its approval by letter ballot of the B18 Committee and the American Society of Mechanical Engineers, the revision was submitted to the American National Standards Institute for recognition as an American National Standard. This was granted on March 23, 1984.

The changes in this 2001 revision follow those made for the metric hex cap screw and include the following:

(a) The government standard items and part numbering system of Appendix III is deleted and replaced by a B18 standard part numbering reference to ASME B18.24.1 found in para. 28.1.

(b) Paragraph 7 makes clear that the head height does not include raised markings.

(c) Bearing face runout is now to be measured on the actual bearing circle instead of at a minimum bearing circle as described in para. 13.

(d) Straightness at maximum material condition is specified. A rail gage replaces the sleeve gage as specified in para. 18.

(e) The transition threads shall have a rounded root contour. This replaces the requirement of a rounded root contour, no radius of which shall be less than the specified minimum at the root of the full form thread for Property Class 10.9 and higher, per para. 20.4.

(f) In Tables 1 and 5 for M16, the head height,  $K$  max., the wrenching height,  $K_w$  min., and the thickness of Gage A,  $T_a$  max., and  $T_a$  min., are each increased by 0.1 mm to agree with ISO/DIS 4162.2.

(g) A new Nonmandatory Appendix A details a comparison with ISO 4162 and ISO/DIS 4162.2.

(h) Dimensional conformance has been specified in para. 26.

(i) Workmanship and accordance with ASTM F 788/F 788M is specified in para. 24.

This Standard was approved by the American National Institute on March 9, 2001.

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*Proposing Revisions.* Revisions are made periodically to the Standard to incorporate changes that appear necessary or desirable, as demonstrated by the experience gained from the application of the Standard. Approved revisions will be published periodically.

The Committee welcomes proposals for revisions to this Standard. Such proposals should be as specific as possible, citing the paragraph number(s), the proposed wording, and a detailed description of the reasons for the proposal, including any pertinent documentation.

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The request for interpretation should be clear and unambiguous. It is further recommended that the inquirer submit his/her request in the following format:

Subject:	Cite the applicable paragraph number(s) and the topic of the inquiry.
Edition:	Cite the applicable edition of the Standard for which the interpretation is being requested.
Question:	Phrase the question as a request for an interpretation of a specific requirement suitable for general understanding and use, not as a request for an approval of a proprietary design or situation. The inquirer may also include any plans or drawings which are necessary to explain the question; however, they should not contain proprietary names or information.

Requests that are not in this format will be rewritten in this format by the Committee prior to being answered, which may inadvertently change the intent of the original request.

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*Attending Committee Meetings.* The B18 Standards Committee regularly holds meetings, which are open to the public. Persons wishing to attend any meeting should contact the Secretary of the B18 Standards Committee.

# METRIC HEX FLANGE SCREWS

## 1 SCOPE

(a) This Standard covers the complete dimensional and general data for metric series hex flange screws recognized as American National Standard.

(b) The inclusion of dimensional data in this Standard is not intended to imply that all products described are stock production items. Consumers should consult with suppliers concerning availability of products.

## 2 COMPARISON WITH ISO 4162

(a) Hex flange screws as presented in this Standard are harmonized to the extent possible with ISO 4162: 1990, and with its draft revision, ISO/DIS 4162.2: 1998. Dimensional differences between this Standard and ISO 4162 are few, relatively minor, and none will affect functional interchangeability of screws manufactured to the requirements of either.

This Standard specifies some requirements that are not included in ISO 4162. **Dimensional requirements shown in bold type are in addition to, or differ from, ISO 4162: 1990 and ISO/DIS 4162.2: 1998.** The technical differences between this Standard and the ISO documents are described in Nonmandatory Appendix A.

(b) Letter symbols designating dimensional characteristics are in accord with ISO 225 and ISO 4162, except where capitals have been used instead of lower-case letters used in the ISO standards.

## 3 REFERENCED STANDARDS

The following is a list of publications referenced in this Standard. Unless otherwise specified, the standard(s) referenced shall be the most recent issue at the time of order placement.

ASME B1.3M Screw Thread Gaging Systems for Dimensional Acceptability—Inch and Metric Threads (UN, UNR, UNJ, M, and MJ)

ASME B1.13M Metric Screw Threads—M Profile

ASME B18.2.8 Clearance Holes for Inch and Metric Bolts and Screws

ASME B18.12 Glossary of Terms for Mechanical Fasteners

ASME B18.18.1M Inspection and Quality Assurance for General Purpose Fasteners

ASME B18.18.2M Inspection and Quality Assurance for High-Volume Machine Assembly Fasteners

ASME B18.24.1 Part Identifying Number (PIN) Code System Standard for B18 Externally Threaded Fasteners

ASME Y14.5M Dimensioning and Tolerancing

Publisher: The American Society of Mechanical Engineers (ASME International), Three Park Avenue, New York, NY 10016-5990; Order Department: 22 Law Drive, Box 2300, Fairfield, NJ 07007

ASTM F 468M Nonferrous Bolts, Hex Cap Screws, and Studs for General Use (Metric)

ASTM F 568M Carbon and Alloy Steel Externally Threaded Metric Fasteners

ASTM B 633 Standard Specification for Electrodeposited Coatings of Zinc on Iron and Steel

ASTM F 738M Stainless Steel Metric Bolts, Screws, and Studs

ASTM F 788/F 788M Surface Discontinuities of Bolts, Screws and Studs, Inch and Metric Series

ASTM F 871M Standard Specification for Electrodeposited Coatings on Threaded Components [Metric] (Discontinued 2001)

ASTM F 1137 Standard Specification for Phosphate/Oil and Phosphate/Organic Corrosion Protective Coatings for Fasteners

Publisher: American Society for Testing and Materials (ASTM), 100 Barr Harbor Drive, West Conshohocken, PA 19428

ISO 225 Fasteners—Bolts, Screws, Studs and Nuts—Symbols and Designations of Dimensions

ISO 898-1 Mechanical Properties of Fasteners Made of Carbon Steel and Alloy Steel—Part 1: Bolts, Screws and Studs

ISO 1891 Bolts, Screws, Nuts and Accessories—Terminology and Nomenclature

ISO 4162 Hexagon Flange Bolts—Small Series

ISO/DIS 4162.2 Hexagon Bolts With Flange—Small Series—Product Grade Combination A/B

ISO 4753 Fasteners—End of Parts With External ISO Metric Thread

ISO 4759-1 Tolerances for Fasteners—Part 1: Bolts, Screws, Studs and Nuts—Product Grades A, B and C

Publisher: International Organization for Standardization (ISO), 1 rue de Varembe, Case Postale 56, CH-1211, Genève 20, Switzerland/Suisse

SAE J1199 Mechanical and Material Requirements for Metric Externally Threaded Steel Fasteners

Publisher: Society of Automotive Engineers (SAE), 400 Commonwealth Drive, Warrendale, PA 15096

#### 4 TERMINOLOGY

For definitions of terms relating to fasteners or features thereof used in this Standard, refer to ASME B18.12.

#### 5 DIMENSIONS

(a) All dimensions in this Standard are given in millimeters (mm), and apply before any coating, unless stated otherwise.

(b) Symbols specifying geometric characteristics are in accord with ASME Y14.5M.

#### 6 TOP OF HEAD

The top of head shall be either full form or indented at manufacturer's option, and shall be either chamfered or rounded. The diameter of the chamfer circle or start of rounding shall be equal to the maximum width across flats,  $S$  maximum, within a tolerance of  $-15\%$ . If the top of the head is indented, the periphery may be rounded.

#### 7 HEAD HEIGHT

The head height,  $K$ , is the distance, parallel to the axis of the screw, from the plane of the bearing circle to the top of the head, not including any raised markings. See para. 22.

#### 8 WRENCHING HEIGHT

The wrenching height,  $K_w$ , is the distance at a corner of the hexagon from the junction of hex head with the flange to the last plane of full-formed hexagon (i.e., the plane closest to the top of the head at which the width across corners,  $E$ , of the hexagon is within its specified limits).

#### 9 CORNER FILL

The rounding due to lack of fill at the six corners of the head shall be reasonably uniform.

#### 10 GAGING OF HEX FLANGE HEAD

See Table 1. The head shall be gaged using two ring gages, A and B, to demonstrate the coincidental acceptability of wrenching height, corner fill, and width across corners. Gage A shall be placed over the head and shall seat on the flange. Gage B shall be placed on the top of the head normal to the screw axis. The two gages shall not be in contact.

#### 11 POSITION OF HEAD

At maximum material condition, the axis of the hexagon of the head shall be within a positional tolerance zone of the diameter specified in Table 2 with respect to the axis of the shank over a distance under the head equal to the nominal screw diameter,  $D$ . The datum shall be as close to the head as practicable, but within  $0.5D$  from the head, and shall be either wholly plain body or wholly the thread pitch diameter, not including the thread runout or the underhead fillet.

#### 12 FLANGE

The top surface of the flange shall be conical or slightly rounded (convex). Radius,  $R_2$ , applies both at the corners and at the flats of the hexagon. The contour of edge at flange periphery, between the maximum flange diameter,  $D_c$  maximum, and the minimum bearing circle diameter,  $D_w$  minimum, shall be optional provided that the minimum flange edge thickness,  $C$  minimum, is maintained at the minimum bearing circle diameter,  $D_w$  minimum.

#### 13 BEARING SURFACE

The bearing surface shall be conical,  $0.75 \pm 0.50$  deg concave from the plane formed by the bearing circle diameter. The plane formed by the bearing circle shall be perpendicular to the axis of the shank, over a length under the head equal to the nominal screw diameter,  $D$ , within the circular runout as specified in Table 2. The measurement of bearing face runout shall be made at the actual bearing circle (i.e., at the line of highest points on any radial line, e.g., by use of straight edge anvil). The datum shall be as close to the head as practical, but within  $0.5D$  from the head, and shall be

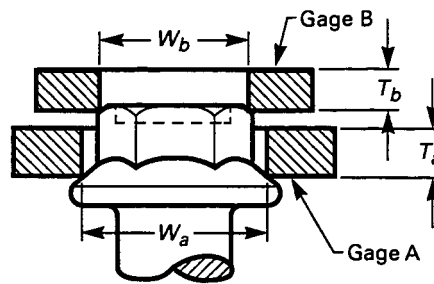


TABLE 1 GAGING OF HEX FLANGE HEAD

Nominal Screw Diameter and Thread Pitch	Gage A				Gage B		
	Inside Diameter, $W_a$		Thickness, $T_a$		Inside Diameter, $W_b$		Thickness, $T_b$
	Max.	Min. [Note (1)]	Max. [Note (2)]	Min.	Max. [Note (3)]	Min.	Min.
M5 × 0.8	8.09	8.08	2.30	2.29	7.43	7.42	3.0
M6 × 1	9.25	9.24	2.90	2.89	8.55	8.54	3.0
M8 × 1.25	11.56	11.55	3.80	3.79	10.79	10.78	4.0
M10 × 1.5	15.02	15.01	4.30	4.29	14.07	14.06	4.0
M12 × 1.75	17.33	17.32	5.40	5.39	16.31	16.30	5.0
M14 × 2	20.79	20.78	5.60	5.59	19.67	19.66	5.0
M16 × 2	24.26	24.25	6.80	6.79	22.57	22.56	6.0

## GENERAL NOTES:

- (a) Dimensions are in millimeters.  
 (b) Refer to para. 10.

## NOTES:

- (1)  $W_a$  min. equals theoretical maximum width across corners,  $E$  max.  
 (2)  $T_a$  max. equals minimum wrenching height,  $K_w$  min.  
 (3)  $W_b$  max. equals minimum width across corners,  $E$  min., -0.01 mm.

TABLE 2 TOLERANCE ZONES

Nominal Screw Diameter and Thread Pitch	Position of Head-to-Shank Tolerance Zone Diameter at MMC	Circular Runout of Bearing Circle to Shank FIM [Note (1)]
M5 × 0.8	0.44	0.16
M6 × 1	0.44	0.20
M8 × 1.25	0.44	0.26
M10 × 1.5	0.54	0.33
M12 × 1.75	0.54	0.39
M14 × 2	0.54	0.46
M16 × 2	0.66	0.53

GENERAL NOTE: Dimensions are in millimeters.

## NOTE:

- (1) Circular runout of bearing circle to shank is based on 1 degree and the minimum bearing circle diameter,  $D_w$  min.

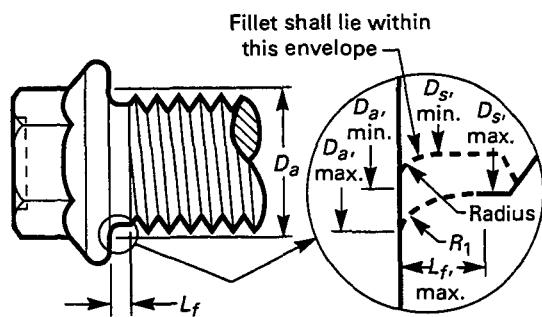
either wholly plain body or wholly thread pitch diameter, not including the thread runout or the underhead fillet.

## 14 FILLET

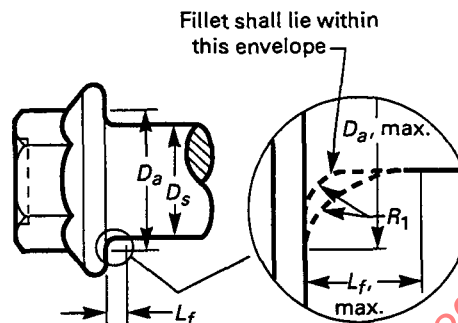
The fillet configuration at the junction of the head and shank shall conform to either Type F, as shown in Table 3, which also specifies limits, or Type U, as shown in Table 4, at the option of the manufacturer unless the fillet type is specified by the purchaser. The fillet shall be a smooth and continuous curve fairing smoothly into the bearing surface and the shank within the limits specified. For Type F, no radius in the fillet contour shall be less than  $R_1$  minimum specified in Table 3.

## 15 BODY DIAMETER

The diameter of the body,  $D_s$ , on screws that are not threaded full length shall be within the limits specified in Table 5, unless the purchaser specifies



(a) Detail for Short Screws



(b) Detail for Long Screws

TABLE 3 DIMENSIONS OF TYPE F UNDERHEAD FILLETS

Nominal Screw Diameter and Thread Pitch	Fillet Transition Diameter, $D_s$		Body Diameter, $D_s$ [Note (2)]	Fillet Length, $L_f$		Fillet Radius, $R_1$
	For Short [Note (1)] and Long Screws	For Short [Note (1)] Screws	For Short [Note (1)] Screws	For Long Screws	For Short [Note (1)] Screws	For Short [Note (1)] and Long Screws
Max.	Min.	Min.	Max.	Max.	Min.	
M5 × 0.8	5.7	5.1	4.36	1.4	0.7	0.2
M6 × 1	6.8	6.2	5.21	1.6	0.9	0.25
M8 × 1.25	9.2	8.3	7.04	2.1	1.1	0.4
M10 × 1.5	11.2	10.2	8.86	2.1	1.2	0.4
M12 × 1.75	13.7	12.2	10.68	2.1	1.3	0.6
M14 × 2	15.7	14.1	12.50	2.1	1.4	0.6
M16 × 2	17.7	16.5	14.50	3.2	1.6	0.6

## GENERAL NOTES:

(a) Dimensions are in millimeters.

(b) Type F was formerly named Style A.

## NOTES:

(1) Short screws are screws that are threaded full length.

(2) Values of  $D_s$  for long screws and  $D_s$  max. for short screws are specified in Table 5.

screws with reduced diameter body. For screws threaded full length, the diameter of the unthreaded shank under the head shall neither exceed the maximum body diameter,  $D_s$  maximum, specified in Table 5 nor be less than the minimum body diameter,  $D_s$  minimum, specified in Table 3 or 4.

Screws of nominal lengths equal to or greater than the shortest nominal lengths specified in Table 6 may be obtained with reduced diameter body, if so specified. Where reduced diameter body (or "Type R") is specified, the body diameter,  $D_2$ , shall be within the limits specified in Table 6. The screw shall have a shoulder under the head. The diameter,  $D_s$ , and length,  $L_2$ , of the shoulder shall be as specified in Table 6.

## 16 LENGTH

The length of the screw is the distance, parallel to the axis of the screw, from the plane formed by the underhead bearing circle diameter to the extreme end of the shank. Tolerances for screw lengths are specified in Table 7.

## 17 POINTS

The end of the screw shall be chamfered or rounded from a diameter equal to or slightly less than the thread root diameter. The length of the point to the first full-formed thread at major diameter, as determined by the

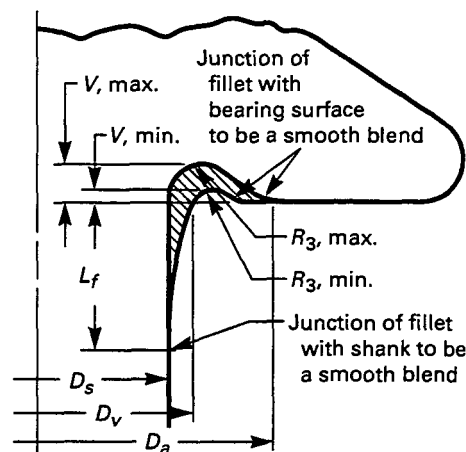


TABLE 4 DIMENSIONS OF TYPE U UNDERHEAD FILLETS

Nominal Screw Diameter and Thread Pitch	Undercut Diameter, $D_u$ [Note (1)]	Fillet Diameter, $D_v$ [Note (2)]	Body Diameter For Short Screws, $D_s$ [Notes (3) and (4)]	Fillet Length, $L_f$	Undercut Radius, $R_3$		Undercut Depth, $V$	
	Max.	Max.	Min.	Max.	Max.	Min.	Max.	Min.
M5 × 0.8	6.2	5.5	4.36	1.4	0.25	0.10	0.15	0.05
M6 × 1	7.5	6.6	5.21	1.6	0.26	0.11	0.20	0.05
M8 × 1.25	10.0	8.8	7.04	2.1	0.36	0.16	0.25	0.10
M10 × 1.5	12.5	10.8	8.86	2.1	0.45	0.20	0.30	0.15
M12 × 1.75	15.2	12.8	10.68	2.1	0.54	0.24	0.35	0.15
M14 × 2	17.7	14.8	12.50	2.1	0.63	0.28	0.45	0.20
M16 × 2	20.5	17.2	14.50	3.2	0.72	0.32	0.50	0.25

## GENERAL NOTES:

(a) Dimensions are in millimeters.

(b) Type U was formerly named Style B.

## NOTES:

(1)  $D_a$  was formerly  $D_u$ .(2)  $D_v$  was formerly  $D_a$ .(3) Values of  $D_s$  for long screws and  $D_s$  max. for short screws are specified in Table 5.

(4) Short screws are screws that are threaded full length.

distance the point enters into a cylindrical NOT GO major diameter ring gage, shall not exceed  $U$  maximum specified in Table 8. The end of the screw shall be reasonably square with the axis of the screw, and where pointed blanks are used, the slight rim or cup resulting from roll threading shall be permissible. At the manufacturer's option, the end of the screw may have a rounded point of radius,  $R_e$ , as specified in Table 8.

## 18 STRAIGHTNESS

At maximum material condition, the derived median line of the screw body and thread major diameter shall

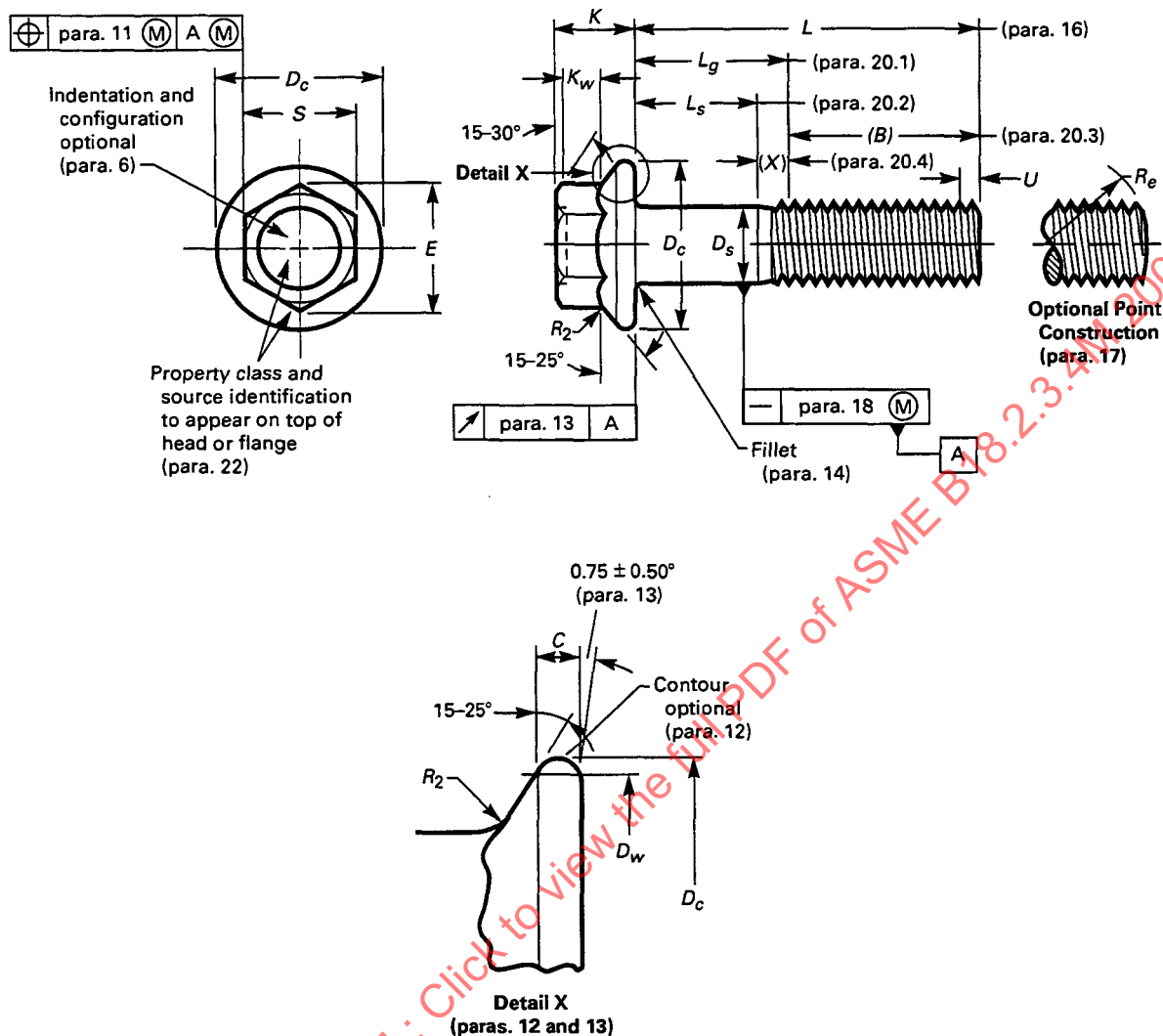
be within a straightness tolerance zone of a diameter equal to **0.006 times length, expressed as a two-place decimal**. A gage and gaging procedure for checking screw straightness are given in Nonmandatory Appendix B.

## 19 THREADS

## 19.1 Thread Series and Tolerance Class

Screw threads shall be general purpose metric screw threads with tolerance Class 6g conforming to ASME B1.13M, unless otherwise specified by the purchaser.





### TABLE 5 DIMENSIONS OF HEX FLANGE SCREWS

Nominal Screw Diameter and Thread Pitch	Body Diameter, $D_s$		Width Across Flats, $S$		Width Across Corners, $E$		Flange Diamter, $D_c$	Bearing Circle Diameter, $D_w$	Flange Edge Thickness, $C$	Head Height, $K$	Wrenching Height, $K_w$	Flange Top Fillet Radius, $R_2$
			Max.	Min.	Max.	Min.						
	Max.	Min.	Max.	Min.	Max.	Min.	Max.	Min.	Min.	Max.	Min.	Max.
M5 × 0.8	5.00	4.82	7.00	6.64	8.08	7.44	11.4	9.4	1.0	5.6	2.3	0.3
M6 × 1	6.00	5.82	8.00	7.64	9.24	8.56	13.6	11.6	1.1	6.9	2.9	0.4
M8 × 1.25	8.00	7.78	10.00	9.64	11.55	10.80	17.0	14.9	1.2	8.5	3.8	0.5
M10 × 1.5	10.00	9.78	13.00	12.57	15.01	14.08	20.8	18.7	1.5	9.7	4.3	0.6
M12 × 1.75	12.00	11.73	15.00	14.57	17.32	16.32	24.7	22.5	1.8	12.1	5.4	0.7
M14 × 2	14.00	13.73	18.00	17.57	20.78	19.68	28.6	26.4	2.1	12.9	5.6	0.9
M16 × 2	16.00	15.73	21.00	20.16	24.25	22.58	32.8	30.6	2.4	15.2	6.7	1.0
See para.	15				8, 9, 10		12	12, 13	12	7, 22	8, 9, 10	12

**GENERAL NOTE:** Dimensions are in millimeters.



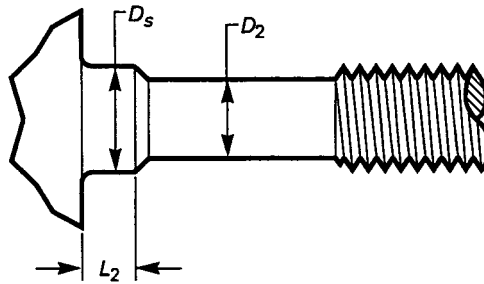


TABLE 6 DIMENSIONS OF REDUCED BODY DIAMETER (TYPE R)

Nominal Screw Diameter and Thread Pitch	Shortest Nominal Screw Length, $L$	Reduced Body Diameter, $D_2$		Shoulder Diameter, $D_s$ [Note (1)]		Shoulder Length, $L_2$ [Note (1)]	
		Max.	Min.	Max.	Min.	Max.	Min.
M5 $\times$ 0.8	30	4.54	4.36	5.00	4.82	3.5	2.5
M6 $\times$ 1	35	5.39	5.21	6.00	5.82	4.0	3.0
M8 $\times$ 1.25	40	7.26	7.04	8.00	7.78	5.0	4.0
M10 $\times$ 1.5	45	9.08	8.86	10.00	9.78	6.0	5.0
M12 $\times$ 1.75	50	10.95	10.68	12.00	11.73	7.0	6.0
M14 $\times$ 2	55	12.77	12.50	14.00	13.73	8.0	7.0
M16 $\times$ 2	60	14.77	14.50	16.00	15.73	9.0	8.0

GENERAL NOTE: Dimensions are in millimeters.

NOTE:

(1) Shoulder is mandatory. See para. 15.

TABLE 7 LENGTH TOLERANCES

Nominal Length, mm	Length Tolerance, mm
Over 6 to 10	$\pm 0.29$
Over 10 to 18	$\pm 0.35$
Over 18 to 30	$\pm 0.42$
Over 30 to 50	$\pm 0.5$
Over 50 to 80	$\pm 0.6$
Over 80 to 120	$\pm 0.7$
Over 120 to 180	$\pm 0.8$
Over 180 to 250	$\pm 0.925$
Over 250 to 300	$\pm 1.05$

For screws with additive finish, size limits for tolerance Class 6g apply before coating, and the thread profile after coating is subject to acceptance using a 6h GO thread gage and tolerance Class 6g thread gage for either minimum material, LO or NOT GO.

## 19.2 Thread Gaging

Unless otherwise specified, dimensional acceptability of screw threads shall be determined based on System 21, ASME B1.3M.

TABLE 8 DIMENSIONS OF POINTS

Nominal Screw Diameter and Thread Pitch	Approximate Point Radius, $R_p$	Max. Point Length, $U$
M5 $\times$ 0.8	7.0	1.6
M6 $\times$ 1	8.4	2.0
M8 $\times$ 1.25	11.2	2.5
M10 $\times$ 1.5	14.0	3.0
M12 $\times$ 1.75	16.8	3.5
M14 $\times$ 2	19.6	4.0
M16 $\times$ 2	22.4	4.0

GENERAL NOTES:

(a) Dimensions are in millimeters.

(b)  $R_p$  equals 1.4 times the nominal screw diameter, and conforms with ISO 4753.(c)  $U$  max. equals two times the thread pitch.

## 20 THREAD LENGTH

The length of thread on screws is controlled by the maximum grip length,  $L_g$  maximum, and the minimum body length,  $L_s$  minimum, as set forth in paras. 20.1 through 20.4.

**TABLE 9 MAXIMUM GRIP LENGTHS,  $L_g$ , AND MINIMUM BODY LENGTHS,  $L_s$** 

Nominal Length, $L$	Nominal Diameter and Thread Pitch													
	M5 × 0.8		M6 × 1		M8 × 1.25		M10 × 1.5		M12 × 1.75		M14 × 2		M16 × 2	
	$L_g$ Max.	$L_s$ Min.	$L_g$ Max.	$L_s$ Min.	$L_g$ Max.	$L_s$ Min.	$L_g$ Max.	$L_s$ Min.	$L_g$ Max.	$L_s$ Min.	$L_g$ Max.	$L_s$ Min.	$L_g$ Max.	$L_s$ Min.
8	1.2	...												
10	2.4	...	1.5	...										
12	2.4	...	3.0	...	1.9	...								
16	2.4	...	3.0	...	4.0	...	2.2	...						
20	2.4	...	3.0	...	4.0	...	4.5	...	2.6	...	3.0	...		
25	9.0	5.0	3.0	...	4.0	...	4.5	...	5.3	...	3.0	...	3.0	...
30	14.0	10.0	12.0	7.0	4.0	...	4.5	...	5.3	...	6.0	...	3.0	...
35	19.0	15.0	17.0	12.0	13.0	6.75	4.5	...	5.3	...	6.0	...	6.0	...
40	24.0	20.0	22.0	17.0	18.0	11.75	14.0	6.5	5.3	...	6.0	...	6.0	...
45	29.0	25.0	27.0	22.0	23.0	16.75	19.0	11.5	15.0	6.25	6.0	...	6.0	...
50	34.0	30.0	32.0	27.0	28.0	21.75	24.0	16.5	20.0	11.25	16.0	6.0	6.0	...
55			37.0	32.0	33.0	26.75	29.0	21.5	25.0	16.25	21.0	11.0	17.0	7.0
60			42.0	37.0	38.0	31.75	34.0	26.5	30.0	21.25	26.0	16.0	22.0	12.0
65					43.0	36.75	39.0	31.5	35.0	26.25	31.0	21.0	27.0	17.0
70					48.0	41.75	44.0	36.5	40.0	31.25	36.0	26.0	32.0	22.0
80					58.0	51.75	54.0	46.5	50.0	41.25	46.0	36.0	42.0	32.0
90							64.0	56.5	60.0	51.25	56.0	46.0	52.0	42.0
100							74.0	66.5	70.0	61.25	66.0	56.0	62.0	52.0
110									80.0	71.25	76.0	66.0	72.0	62.0
120									90.0	81.25	86.0	76.0	82.0	72.0
130											90.0	80.0	86.0	76.0
140											100.0	90.0	96.0	86.0
150													106.0	96.0
160													116.0	106.0

**GENERAL NOTES:**

- (a) Dimensions are in millimeters.  
 (b) Diameter-length combinations between the thin stepped lines are recommended.  
 (c) Screws with lengths above the solid thick stepped lines are threaded full length. See Table 10.  
 (d) For screws with longer lengths,  $L_g$  and  $L_s$  values shall be computed from formulas as given in para. 20.

**20.1 Grip Length,  $L_g$** 

The grip length,  $L_g$ , is the distance, measured parallel to the axis of the screw, from the plane of the bearing circle diameter to the face of a noncounterbored, non-countersunk GO thread ring gage assembled by hand as far as the thread will permit. For standard diameter-length combinations of screws, the values for  $L_g$  maximum are specified in Table 9. For diameter-length combinations not listed in Table 9, the maximum grip length for long screws that are not threaded full length is equal to the nominal screw length,  $L$ , minus the

reference thread length,  $(B)$ , as specified in Table 10:  
 $L_g \text{ maximum} = L \text{ nominal} - (B)$ .

For short screws of nominal lengths,  $L$ , that are shorter than the lengths specified in Table 10 for screws threaded full length,  $L_g \text{ maximum} = A \text{ maximum}$  as specified in Table 10.

**20.2 Body Length,  $L_s$** 

Body length,  $L_s$ , on long screws that are not threaded full length is the distance, parallel to the axis of the screw, from the plane of the bearing circle to the last

TABLE 10 THREAD LENGTHS

Nominal Screw Diameter and Thread Pitch	Thread Length, (B, Ref.)			Transition Thread Length (X, Ref.)	Screws Threaded Full Length				
	Screw Length, L				Screw Length, L	Unthreaded Length Under Head, A		Unthreaded Length Under Head, A	
	≤125	>125 and ≤200	>200			At Least	Under	Max.	
M5 × 0.8	16	22	35	4.0	10	1.2	10	25	2.4
M6 × 1	18	24	37	5.0	12	1.5	12	30	3.0
M8 × 1.25	22	28	41	6.25	16	1.9	16	35	4.0
M10 × 1.5	26	32	45	7.5	20	2.2	20	40	4.5
M12 × 1.75	30	36	49	8.75	24	2.6	24	45	5.3
M14 × 2	34	40	53	10.0	28	3.0	28	50	6.0
M16 × 2	38	44	57	10.0	32	3.0	32	55	6.0

GENERAL NOTE: Dimensions are in millimeters.

scratch of thread or top of the extrusion angle, whichever is closer to the head. For standard diameter-length combinations of screws, the values for  $L_g$  minimum are specified in Table 9. For diameter-length combinations not listed in Table 9, the minimum body length on screws that are not threaded full length is equal to the maximum grip length, as determined above, minus the transition thread length, (X), as specified in Table 10:  $L_g$  minimum =  $L_g$  maximum - (X).

### 20.3 Thread Length, (B)

The thread length, (B), specified in Table 10, is a reference dimension intended for calculation purposes only, and is the distance, parallel to the axis of the screw, from the extreme end of the screw to the last complete (full-form) thread.

### 20.4 Transition Thread Length, (X)

The transition thread length, (X), specified in Table 10, is a reference dimension intended for calculation purposes only. It includes the length of incomplete threads and tolerances on grip length and body length. The transition from full-form thread to incomplete thread shall be smooth and uniform. The major diameter of the incomplete threads shall not exceed the actual major diameter of the complete (full-form) threads. The transition threads shall have a rounded root contour.

## 21 MATERIALS AND MECHANICAL PROPERTIES

### 21.1 Steel

Unless otherwise specified, steel screws shall conform to the requirements for Property Class 9.8 or Property Class 10.9 as specified in ASTM F 568M, or SAE J1199.

### 21.2 Corrosion-Resistant Steels

Unless otherwise specified, corrosion-resistant steel screws shall conform to the requirements of ASTM F 738M.

### 21.3 Nonferrous Metals

Unless otherwise specified, nonferrous screws shall conform to the requirements of ASTM F 468M.

## 22 IDENTIFICATION SYMBOLS

Markings shall be raised or recessed on the top of the head or raised on the top of the flange unless otherwise specified by the purchaser. Markings shall be legible to the unaided eye with the exception of corrective lenses. When raised, markings shall project not less than 0.1 mm for M14 and smaller screws, and 0.3 mm for M16 screws, above the top surface of the head or flange; and total head height (head plus markings) shall not exceed the specified maxi-

imum head height,  $K$  maximum, plus 0.1 mm for M5 and M6 screws, 0.2 mm for M8 and M10 screws, 0.3 mm for M12 and M14 screws, and 0.4 mm for M16 screws.

## 22.1 Property Class Symbols

Each screw shall be marked in accordance with the requirements of the applicable specification for its material and mechanical properties.

## 22.2 Source Symbols

Each screw shall be marked to identify its source (manufacturer or private label distributor).

## 23 FINISH

Unless otherwise specified, screws shall be supplied with a natural (as processed) finish, unplated or uncoated, in a clean condition, and lightly oiled.

## 24 WORKMANSHIP

Screws shall be free from surface imperfections such as burrs, seams, laps, loose scale, and other surface irregularities that could affect serviceability and shall conform to ASTM F 788/F 788M.

## 25 INSPECTION AND QUALITY ASSURANCE

Unless otherwise specified, acceptability of screws shall be determined in accordance with ASME B18.18.1M.

## 26 DIMENSIONAL CONFORMANCE

Products shall conform to the specified dimensions. Unless otherwise specified, the following provisions shall apply for inspection of dimensional characteristics:

(a) Unless otherwise specified, the following designated dimensional characteristics shall be inspected to the inspection levels shown according to ASME B18.18.2M, and shall be within their specified limits.

Characteristic	Inspection Level
Thread acceptability	C
Body diameter, $D_s$	C
Gaging of hex flange head	C
Grip length, $L_g$ max.	C
Screw length, $L$	C
Visual inspection [Note (1)]	C

### NOTE:

(1) Visual inspection shall include property class marking, source marking, fillet, and workmanship.

If verifiable in-process inspection is used, inspection sample sizes and reporting shall be in accordance with the applicable ASME, ASTM, or SAE quality system consensus standard.

(b) For nondesignated dimensional characteristics, the provisions of ASME B18.18.1M shall apply. Should a nondesignated dimension be determined to be outside its specified limits, it shall be deemed conforming to this Standard if the user, who is the installer, accepts the dimension, based on form, fit, and function considerations.

## 27 CLEARANCE HOLES

The recommended sizes of clearance holes in material to be assembled using hex flange screws are the normal series given in ASME B18.2.8.

## 28 DESIGNATION

(a) Hex flange screws shall be designated by the following data, preferably in the sequence shown: product name and designation of the standard, nominal diameter and thread pitch, nominal length, steel property class or material identification, and protective coating, if required.

### EXAMPLES:

(1) Hex flange screw B18.2.3.4M, M10 × 1.5 × 50, Class 9.8, zinc plated per ASTM F 1941M and ASTM B 633 Fe/Zn 5 Type II.

(2) Hex flange screw B18.2.3.4M, M6 × 2 × 100, Class 10.9, phosphate/oil, ASTM F 1137 Grade I.

(3) Hex flange screw B18.2.3.4M, reduced diameter body, fillet Type U, M16 × 2 × 80, Class 10.9 oiled.

(4) Hex flange screw B18.2.3.4M, M16 × 1 × 30, A1-70 ASTM F 738M.

NOTE: It is common practice in ISO standards to omit thread pitch from the nominal size designation when screw threads are the metric coarse thread series; e.g., M10 is M10 × 1.5.

(b) For a recommended part identifying number (PIN) system for metric hex flange screws, see ASME B18.24.1.